

Express Mail No. EV178023293US

PATENT APPLICATION OF
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ENTITLED
FLOW METER FOR USE WITH HIGH PRESSURE
PROCESS FLUID

Docket No. **R11.12-0853**

FLOW METER FOR USE WITH HIGH PRESSURE

PROCESS FLUID

BACKGROUND OF THE INVENTION

The present invention relates to a vortex flow meter for measuring fluid flow, and more particularly, a flow meter for use with high pressure process fluid.

Various differential pressure sensitive vortex flow meters have been advanced, which operate on a principle that a buff body or shedding bar placed in a fluid flow causes or generates vortices alternately on opposite sides of the shedding bar, causing variations in pressure on either side of the bar. The frequency of vortex shedding for an individual bar configuration characteristic is directly proportional to the velocity of flow in the stream.

Vortex flow meters are known in the prior art, and examples of vortex flow meter implementations can be found in U.S. Patent No. 4,926,695 issued to Kleven et al. on May 2, 1990, U.S. Patent No. 5,343,762 issued to Beulke on September 6, 1994, which are incorporated herein by reference.

Typically, a vortex flow meter for measuring fluid flow includes a conduit having a wall surrounding a bore for carrying the fluid. The wall has a wall region of reduced thickness formed therein. The wall region of reduced thickness is sometimes referred to as a "region of reduced

stiffness" or a "flexure". A shedding bar is disposed in the bore. In a typical embodiment, the shedding bar includes an upstream extremity, a downstream extremity and an intermediate portion connecting the upstream and downstream extremities. The intermediate portion includes a region of reduced stiffness which flexes in response to disturbances or vortices within the fluid created by fluid flow around the upstream extremity to promote motion of at least a portion of the downstream extremity. Such flow meters further include sensing means coupled to the downstream extremity for sensing the motion and providing an output as a function thereof. Generally, the sensing means senses lateral motion, and is removably attached to a post extending from the wall region away from the bore, wherein the post transmits the motion to the sensing means.

One method of assuring that a vortex meter meets process pressure retention (strength) requirements is described in the American Society of Mechanical Engineers (ASME) Boiler Pressure Vessel Code (BPVC). To determine the factor of safety the meter has at a given pressure, a meter representative of the design is pressurized until the structure bursts. Factors are derived from testing of the first meter body produced, and calculations are performed based on various factors of the material utilized to make the device. For example, material characteristics, including composition and

manufacturing processes, are factored into a calculation to determine the maximum pressure to which a device may be rated.

For higher pressures, the flexure often 5 presents a weak point in the structure, which can tear or burst open when exposed to higher pressures. Since the flexure must be designed to be thin enough to permit movement of the post that is coupled to the buff body or shedding bar, conventional flexures have 10 difficulting providing the required safety factor at high pressures.

SUMMARY

A vortex flow meter for measuring fluid flow has a conduit for carrying a fluid. A region of 15 reduced thickness is formed in a portion of the conduit. A shedder bar disposed in the conduit is coupled to the region of reduced thickness and is configured to apply a rocking motion to the region of reduced thickness about a pivot line in response to 20 flow of the fluid. At least one reinforcing rib on the reduced thickness portion extends parallel to flow of the fluid.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a partial cut away view of a flow 25 meter in situ according to an embodiment of the present invention.

FIG. 2 is an enlarged view of the flexure portion of the flow meter of FIG. 1.

FIG. 3 is a cross-sectional view of a flow meter taken along line 3-3 in FIG. 1.

FIG. 4 is an expanded view of the region of reduced thickness taken in cross-section along line 5 3-3 in FIG. 1.

FIG. 5 is a top view of the region of reduced thickness of FIG. 1.

DETAILED DESCRIPTION

In order to qualify for use in a particular 10 pressure rating, users are required to withstand a burst pressure test that gives effective safety factor according to the ASME BPVC.

$$\text{Burst} > (\text{safety factor}) \times \frac{PR}{(FxR)}$$

where PR is the maximum operating pressure for a 15 pressure rating, F is a casting quality factor, and R is the ratio of actual to minimum tensile strength of the material utilized in the particular implementation. The safety factor may depend on the specific usage and the implementation.

The present invention includes a 20 modification to conventional flexures (sometimes referred to as regions of reduced wall thickness) to include a rib to extend across the flexure in the direction of flow on either side of the center line 25 of the flexure. Additionally, the size of a center rib disposed on the center line of the flexure is increased. With these enhancements, the flexure is able to retain (withstand) a pressure that is

approximately 150% that of a conventional meter. Additionally, the side ribs increase the sensitivity of the meter to alternating pressure created by the shedding bar, by reducing the countervailing force on 5 the sensor due to pressure on the rigid center flexure of the meter.

Figure 1 illustrates an embodiment of a vortex flow meter of the present invention. Flowmeter 10 includes a conduit 12 having a wall 14 surrounding a bore 16. Bore 16 carries a fluid, which may be a liquid or a gas, generally along a bore axis 18. Shedding bar 20 is a vortex-generating obstruction. Pivoting member 22 extends from a hole 24 formed in wall 14 into bore 16. Fluctuating fluid 10 pressures act on the shedding bar and on pivoting member 22, such that pivoting member 22 moves in 15 response to the fluctuating pressures.

A center rib 26 is disposed in hole 24 and coupled to flexure 28. The flexure 28 is generally 20 coupled to wall 14. Post 30 extends from the flexure 28. A sensing device 32 (shown in phantom) is coupled to the flexure 28, preferably by attachment to the post 30, and senses the motion of pivoting member 22. The sensing device 32 generates an output 25 indicative of the sensed motion and communicates the output via lead 34 to circuitry 36. Typically, circuitry 36 is adapted to communicate the sensed motion to a control center 38 via a communications link 40 (which may be a two-wire, three-wire or four-

wire loop, or which may be a wireless communications link 40).

In a preferred embodiment, reinforcing ribs 42 are disposed on the flexure on either side of centerline 27 (or pivot line) of the flexure 28 (reference numeral 27 is shown in FIGS. 2 and 5). In general, the center rib 26 and the reinforcing ribs 42 extend parallel to the direction to the bore axis 18 (parallel to the direction of flow).

10 In one embodiment, the reinforcing ribs 42 are positioned on the flexure between a center of the center rib 26 and the outside edge of the flexure 28. The position may be quantified by considering a flexure 28 of diameter D, where the reinforcing ribs 15 42 are positioned at approximately 0.25D on either side of the center rib 26. In an alternative embodiment, the reinforcing ribs 42 are centered between the edge of the center rib 26 and the outside edge of the flexure 28 (at approximately 0.22D, for 20 example, relative to the edge of the flexure 28). In this embodiment, the center rib 26 is approximately the same width as the post 30 and extends the entire diameter of the flexure 28. In an alternative embodiment, the center rib 26 is a fraction of an 25 inch wider than the post 30 and extends the full diameter of the flexure. The reinforcing ribs are approximately the same relative height and width. In one embodiment, the reinforcing ribs have a height and a width that are approximately the same as the

thickness of the flexure. It will be understood by a worker skilled in the art that the flexure 28 is curved so as to match the wall of the pipe. Thus, the height of the reinforcing ribs 42 is relative to 5 the surface of the flexure 28, but varies relative to a fixed point according to a radius of curvature of the pipe.

Depending on the implementation, the rib size may be adjusted to account for different 10 pressures and different flexure sizes. Additionally, the location of the reinforcing ribs 42 may be adjusted toward or away from the center line. In a preferred embodiment, the reinforcing ribs 42 are substantially centered between the center line of the 15 flexure 28 and the outer edge of the flexure 28.

It is appreciated that these reinforcing ribs 42 add to the burst pressure strength of the meter. Adding the reinforcing ribs 42 to a meter typically increases the burst pressure of the meter 20 by 50%. Additionally, the reinforcing ribs 42 increase the sensitivity of the flow meter to the alternating pressure from the fluidic vortices, by reducing the countervailing force on the sensor due to pressure on the flexure 28 of the meter 10.

25 FIG. 2 shows an expanded view of the flexure 28 of FIG. 1. As shown, the conduit 12 has an opening 24 in which a flexure 28 is disposed. Center rib 26 extends a full diameter of the flexure, and is positioned on a center line 44 of the flexure

28. In this embodiment, reinforcing ribs 42 are disposed on either side of the center rib 26 and extend to the outer edge of the flexure 28. The reinforcing ribs 42 are positioned at approximately a 5 midpoint between the center line 44 and the outer edge of the flexure 28. Post 30 extends from the center rib 26 and away from the conduit 12. As previously discussed, in a preferred embodiment, the sensor is coupled to the shedding bar via the post 10 30.

FIG. 3 shows a cross sectional view of the vortex flow meter 10 of FIG. 1 taken along line 3-3. Flowmeter 10 includes a conduit 12 having a wall 14 surrounding a bore 16. Bore 16 carries a fluid, 15 which may be a liquid or a gas, generally along a bore axis 18. Shedding bar 20 is a vortex-generating obstruction. Pivoting member 22 (shown in FIG. 1) extends from the wetted side of the flexure 28. Fluctuating fluid pressures act on the shedding bar 20 and on pivoting member 22, such that pivoting member 22 moves in response to the fluctuating pressures.

In this embodiment, center rib 26 is disposed in hole 24 and coupled to flexure 28. The flexure 28 is generally coupled to wall 14. Post 30 25 extends from the flexure 28. Reinforcing ribs 42 are disposed on the flexure on either side of the center rib 26 on either side of a centerline of the flexure 28. In general, the center rib 26 and the reinforcing ribs 42 extend parallel to the direction

to the bore axis 18 (parallel to the direction of flow). Flange element 46 couples a transmitter housing 48 (shown in phantom) to the meter body. The transmitter housing 48 may also contain circuitry for 5 processing sensed data into a signal for transmission to a control center.

FIG. 4 illustrates an expanded cross-sectional view of the flexure 28. Flexure 28 is disposed in opening 24 within conduit 12. Center rib 10 26 is centered on the flexure 28, and reinforcing ribs 42 are approximately centered between the center rib 42 and the outer circumferential edge of the flexure 28. The center rib 26 and the reinforcing ribs 42 extend in parallel to each other and to the 15 direction of fluid flow. Pivoting member 22 is disposed on shedding bar 20 and is coupled to the post 30 to cause the post 30 to move responsive to fluidic pressures experienced by the pivoting member 22 and the shedding bar 20.

20 In general, the flexure 28 (region of reduced thickness) has a thickness T. The reinforcing ribs 42 have a width W and a height H. It should be understood that the flexure 28 is curved to match the curvature of the pipe wall, such that 25 the height H of the reinforcing ribs 42 is relative to the curved surface of the flexure 28. In a preferred embodiment, the width W and the height H are approximately equal to the thickness T.

FIG. 5 shows the flexure 28 from a top view looking into opening 24. The flexure 28 has a diameter D, which can be measured from edge to edge along the outer (non-wetted) surface of the flexure 28. In one preferred embodiment, the position of the reinforcing ribs 42 may be determined relative to the centerline (pivot line) 27 of the flexure 28, such that the reinforcing ribs 42 are centered at approximately 0.25D.

As previously discussed, the primary method of determining the pressure rating for a meter is a burst pressure test. In these tests, the flow meter is connected to a testing device and then pressurized until the meter fails because it can no longer retain the pressure. In all test cases, failures occurred in the flexure area, which is the thinnest wall section.

For the purposes of the ASME code, the safety factor for a meter is determined by the following equation:

$$SF = \frac{F \times BP}{P_r} \times \frac{T_s}{T_a}$$

where SF is the safety factor, F is a casting quality factor, BP is burst pressure, P_r is rated pressure, T_s is the material's tensile strength specification, and T_a is actual tensile strength of the material tested as determined by a test specimen from the heat the casting is taken from.

Adding a reinforcing rib 42 to the flexure 28 between the centerline and the edge of the flexure 28 significantly increases the ultimate burst pressure. In one example test, the reinforcing ribs 5 were approximately 0.035 inches wide and 0.070 inches tall and extended parallel to the center rib 26. The reinforcing ribs 42 were positioned at the area where the maximum total displacement occurred in the prior art design. Calculations predicted a greater than 30 10 percent increase in burst pressure. The tested device showed improvement in burst pressure of 50 percent.

Further analysis of the reinforced flexure design showed improved sensitivity over conventional flexure designs. Calculations predicted an increase 15 in sensitivity of approximately 6 percent. In testing, the reinforcing ribs 42 increased sensitivity of the meter up to 20 percent in some embodiments.

It is also important to note that the 20 center rib contributes to a restorative force tending to restore pivoting member 22 to its equilibrium position. The restoring force tends to increase the natural frequency of vibration of the pivoting member, which ensures that the natural frequency of 25 the pivoting member 22 is greater than the highest vortex frequency encountered during operation. The reinforcing ribs may actually improve this restorative force.

Additionally, it should be noted that in the embodiments shown, the reinforcing ribs are offset from a pivot line of the region of reduced thickness. In a preferred embodiment, the 5 reinforcing ribs are approximately centered between a pivot line and an edge of the region of reduced thickness, making the offset distance approximately the same. However, in some embodiments, it may be desirable to vary the position of the reinforcing 10 ribs. For example, the position of the reinforcing ribs may be offset from the midpoint between the pivot line and the edge, by approximately the same distance. Alternatively, the reinforcing ribs may be spaced from the pivot line by slightly different 15 distances so as to locate the reinforcing rib in a particular location on the region of reduced thickness.

Finally, while the above-discussion has largely described the reinforcing ribs as extending 20 substantially parallel to the pivot line (or center rib), in some embodiments, it may be desirable to change the angle of reinforcing ribs relative to the center line. For example, in one embodiment, it may be desirable to position the reinforcing rib on the 25 flexure adjacent to the pivot line and extending at an angle from the pivot line. In such an embodiment, the position of the reinforcing rib relative to the pivot line or the circumferential edge would vary depending on where the measurement was taken. By

changing the angle, it may be possible to adjust the sensitivity of the meter while maintaining the improved pressure retention advantages.

Although the present invention has been described with reference to preferred embodiments, workers skilled in the art will recognize that changes may be made in form and detail without departing from the spirit and scope of the invention.